

The trace-determinant plane

There is a nice geometric object called the trace-determinant plane that organizes the various types of 2×2 linear systems.

Consider the 2×2 matrix

$$\mathbf{A} = \begin{pmatrix} a & b \\ c & d \end{pmatrix}.$$

Let's calculate the characteristic polynomial of \mathbf{A} :

Conclusion: The eigenvalues of any 2×2 matrix are determined by the trace and the determinant of \mathbf{A} . We have

$$\lambda = \frac{(\operatorname{tr} \mathbf{A}) \pm \sqrt{(\operatorname{tr} \mathbf{A})^2 - 4(\det \mathbf{A})}}{2}.$$

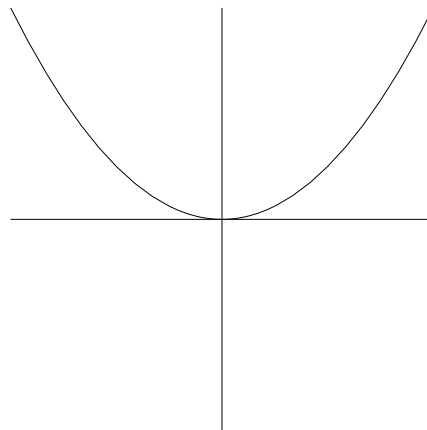
Summary of Phase Portraits

Assume $\det \mathbf{A} \neq 0$. Then zero is not an eigenvalue of \mathbf{A} .

1. Real and distinct eigenvalues
 - (a) sink
 - (b) saddle
 - (c) source
2. Complex eigenvalues
 - (a) spiral sink
 - (b) center
 - (c) spiral source
3. Real and repeated eigenvalues
 - (a) sink with one eigenline in the phase portrait
 - (b) source with one eigenline in the phase portrait
 - (c) sink where every solution is a straight-line solution
 - (d) source where every solution is a straight-line solution

What if $\det \mathbf{A} = 0$?

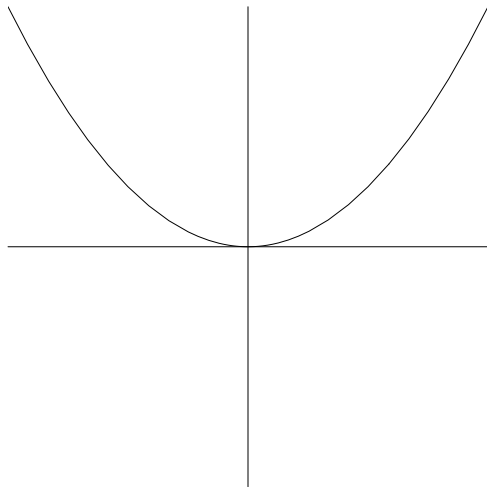
We can organize these different types using a plane with unusual coordinate axes.



You can turn on the trace-determinant plane in the `LinearPhasePortraits` tool.

Example. Consider the one-parameter family of linear systems

$$\frac{d\mathbf{Y}}{dt} = \begin{pmatrix} -1 & 1 \\ 0 & d \end{pmatrix} \mathbf{Y}.$$



Forced equations

For the last five weeks, all of our differential equations have been autonomous. Now we turn to second-order equations that model systems that are subject to some type of external forcing. Here are two examples:

Example. The nonlinear pendulum with a pivot point that is subject to vertical oscillations. The motion of such a pendulum is governed by the second-order nonlinear equation

$$m \frac{d^2\theta}{dt^2} + b \frac{d\theta}{dt} + k \sin \theta = F \sin \theta \cos \omega t$$

where ω determines the frequency of the oscillations of the pivot point and F determines the amplitude of the oscillations. The `Pendulums` tool on the CD illustrates this system.

Example. The linear mass-spring system where the spring is subject to vertical oscillations. To model this system, we use the standard mass-spring system and add a term that corresponds to the force added to the system by the oscillations. We get

$$m \frac{d^2y}{dt^2} + b \frac{dy}{dt} + ky = F \cos \omega t.$$

The `ForcedMassSpring` tool on the CD illustrates this system.

In class we will discuss forced linear equations only, but your second project will involve some experimentation with a forced nonlinear system.